

8-79. The joint is used to transmit a torque of $T = 2.4 \text{ kN}\cdot\text{m}$. Determine the required nominal diameter of the shaft pin A if it is made from a material having a shear failure stress of $\tau_{\text{fail}} = 150 \text{ MPa}$. Apply a factor of safety of 3 against failure.



Relevant Knowledge: The shear force developed on the shear plane of pin A can be determined by writing the moment equation of equilibrium along the x axis with reference to the free-body diagram of the shaft, Fig. a.

$$\sum M_x = 0; \quad 4(10)(1) - 5(10)^2 = 0$$

$$4 = 5(10)^2$$

Allowable Shear Stress:

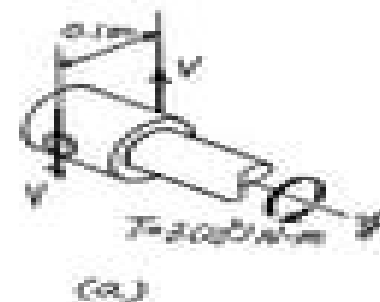
$$\tau_{\text{allow}} = \frac{\tau_{\text{fail}}}{FS} = \frac{150}{3} = 50 \text{ MPa}$$

Using this result,

$$\tau_{\text{allow}} = \frac{T}{A_s}; \quad 50(10^6) = \frac{2.4(10^3)}{\frac{\pi}{4} d_s^2}$$

$$d_s = 0.00794 \text{ m} = 7.94 \text{ mm}$$

Ans.



8-80. Determine the maximum allowable torque T that can be transmitted by the joint. The shear pin A has a diameter of 25 mm, and it is made from a material having a failure shear stress of $\tau_{\text{fail}} = 150 \text{ MPa}$. Apply a factor of safety of 3 against failure.



Relevant Knowledge: The shear force developed on the shear plane of pin A can be determined by writing the moment equation of equilibrium along the x axis with reference to the free-body diagram of the shaft, Fig. a.

$$\sum M_x = 0; \quad 4(10)(1) - T = 0$$

$$4 = T$$

Allowable Shear Stress:

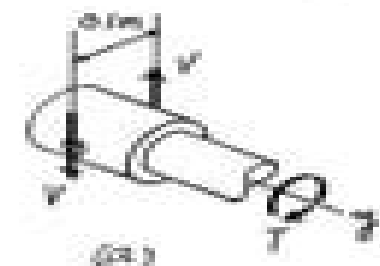
$$\tau_{\text{allow}} = \frac{\tau_{\text{fail}}}{FS} = \frac{150}{3} = 50 \text{ MPa}$$

The area of the shear plane for pin A is $A_s = \frac{\pi}{4} (25)^2 = 490.87 \text{ mm}^2$. Using this result,

$$\tau_{\text{allow}} = \frac{T}{A_s}; \quad 50(10^6) = \frac{T}{490.87(10^{-6})}$$

$$T = 24543.5 \text{ N}\cdot\text{m} = 24.5 \text{ kN}\cdot\text{m}$$

Ans.



Mechanics Of Materials Hibbeler 8th Edition Solution Manual Slideshare

R Barnett



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