

To calculate ρ_{air} use the following relation

$$\rho_{air} = \frac{1.01325 \times 10^5}{287 \times (273 + T_a)} \text{ in Kg/m}^3$$

$$3. \text{ Brake Horse Power (BHP)} = \frac{V \times I}{1000 \times \eta_g} \text{ in KW}$$

Where,

V = Voltmeter reading

I = Ammeter reading

η_g = Efficiency of Generator = 0.75

$$4. \text{ Air - Fuel Ratio} = \frac{m_a}{m_f}$$

$$5. \text{ Heat input } Q = m_f \times C V \text{ in kW}$$

where m_f is the mass of fuel supplied in kg/s

C V is the calorific value of the fuel in kJ/kg

$$6. \text{ Specific fuel consumption (SPC)} = \frac{m_f \times 3600}{BP} \text{ in kg/kWhr}$$

$$7. \text{ Volumetric efficiency } \eta_{vol} = \frac{\text{actual vol of air supplied } V_{act}}{\text{Theoretical air supply } V_{th}} \times 100$$

Theoretical air supply V_{th} = Swept volume = $L \times A \times N$ in m^3/min

where $A = \pi D^2/4$ in m^2 L and D are the stroke and bore of the engine.

V_{act} = Actual Volume of air supplied in m^3/min

$$8. \text{ Brake thermal efficiency } \eta_{bth} = \frac{BP}{\text{Heat input } Q} \times 100$$

HEAT BALANCE SHEET:

Heat input	KW	in %	Heat Output	KW	in %
Heat supplied by the fuel			a) Heat equivalent to BP b) Heat carried by exhaust gases = $m_g \times C_{pg} (T_g - T_a)$ $m_g = m_a + m_f$ c) Heat unaccountable 1-(a+b)		
Total input			Total output		

Specific heat of air = $C_{pg} = 1.005 \text{ KJ/Kg}^\circ\text{C}$

m_a = Mass of air supplied in Kg/s

m_f = mass of fuel supplied in Kg/s

Energy Conversion Engineering Lab Manual

Valentine Korah



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