

M/C DESIGN

- Length of open belt drive (O.B.D)

$$= 2C + \frac{\pi}{2}(D_1 + D_2) + \frac{(D_2 - D_1)^2}{4C}$$

- Length of cross belt driven (C.B.D)

$$= 2C + \frac{\pi}{2}(D_1 + D_2) + \frac{(D_2 + D_1)^2}{4C}$$

- Velocity ratio

$$V.R = \frac{N_2}{N_1} = \left(\frac{D_1 + t}{D_2 + t} \right) \left(1 - \frac{S}{100} \right)$$

- Ratio of belt tension

$$\frac{T_1}{T_2} = e^{\mu \theta}$$

- Maximum tensile strength of belt

$$T_{\max} = b.t.\sigma_{\text{per}}$$

- Power transmission capacity of belt drive

$$P.T.C = (T_1 - T_2)v$$

- Centrifugal tension

$$T_c = \frac{T_{\max}}{3}$$

- Initial tension

$$T_0 = \frac{T_1 + T_2 + 2T_c}{2}$$

- Number of 'V' belts

$$n = \frac{P_{\text{total}}}{P_{\text{each}}} \times K_q$$

- Buckingham dynamic load

$$F_d = F_1 + \frac{20.67v[bc + F_1]}{20.67v + \sqrt{bc + F_1}} \quad c = \frac{a}{K \left(\frac{1}{E_1} + \frac{1}{E_2} \right)}$$

- Soderberg equation

$$\frac{1}{FOS} = \frac{\sigma_m}{\sigma_y} + \frac{K_f \sigma_v}{\sigma_s}$$

- Goodman equation

$$\frac{1}{FOS} = \frac{K_1 \sigma_m}{\sigma_{ut}} + \frac{K_f \sigma_v}{\sigma_e}$$

- Notch sensitivity index

$$q = \frac{K_f - 1}{K_t - 1}$$

- Unwin's formula

$$d = 6\sqrt{t}$$

- Area of transverse fillet weld

$$A_{T.F.W} = h\ell_s \quad h = \frac{t}{\cos \theta + \sin \theta}$$

- Strength of T.F.W = $0.832t\ell_s\tau_{\text{per}}$

- Strength of parallel fillet weld

$$= 0.707t\ell_s\tau_{\text{per}}$$

- Strength of Butt weld = $h\ell(\sigma_t)_{\text{per}}$

- Frictional torque of thrust bearing by uniform pressure theory

$$(T_f)_{\text{UPT}} = \frac{2}{3}\mu\omega \left(\frac{R_o^3 - R_i^3}{R_o^2 - R_i^2} \right)$$

- Frictional torque of thrust bearing by uniform wear theory

$$(T_f)_{\text{UWT}} = \mu\omega \left(\frac{R_o + R_i}{2} \right)$$

- Power loss $P_{\text{loss}} = T_f.\omega$

- T_f equations for flat pivot bearing

$$(T_f)_{\text{UPT}} = \frac{2}{3}\mu WR$$

$$(T_f)_{\text{UWT}} = \frac{1}{2}\mu WR$$

- Bearing pressure

$$p_{\text{ind}} = \frac{W}{LD}$$

- Strength of bearing = $p_{\text{per}} \times L \times D$

- Bearing characteristic number = $\frac{zn}{p}$

- Mc-Kee's equation

$$\mu = \frac{33}{10^8} \left[\left(\frac{zn'}{p'} \right) \left(\frac{D}{C} \right) \right] + K$$

- Somer field No.

$$S = \left(\frac{zn}{p} \right) \left(\frac{D}{C} \right)^2$$

- Petroff's equation for μ

$$\mu = \frac{2\pi^2}{p} \left(\frac{zn}{p} \right) \left(\frac{D}{C} \right)$$

- Life of bearing

$$L_{50} = \left(\frac{C}{P_e} \right)^k \quad (\text{Million rev.})$$

K_q = service or overload factor.

Mechanical Engineering Design And Formulas For Manufacturing

Ying-Ying Zheng



Mechanical Engineering Design And Formulas For Manufacturing:

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In an electronic era where connections and knowledge reign supreme, the enchanting power of language has become more apparent than ever. Its capability to stir emotions, provoke thought, and instigate transformation is truly remarkable. This extraordinary book, aptly titled "**Mechanical Engineering Design And Formulas For Manufacturing**," written by a highly acclaimed author, immerses readers in a captivating exploration of the significance of language and its profound effect on our existence. Throughout this critique, we shall delve into the book's central themes, evaluate its unique writing style, and assess its overall influence on its readership.

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