

8-76. The joint is used to transmit a torque of $T = 2.4 \text{ kN}\cdot\text{m}$. Determine the required nominal diameter of the shaft pin A if it is made from a material having a shear failure stress of $\tau_{\text{fail}} = 150 \text{ MPa}$. Apply a factor of safety of 3 against failure.



Relevant Knowledge: The shear force developed on the shear plane of pin A can be determined by writing the moment equation of equilibrium along the x axis with reference to the free-body diagram of the shaft, Fig. a.

$$\sum M_x = 0; \quad (150)(L) - 500(5) = 0$$

$$L = 16.67 \text{ mm}$$

Allowable Shear Stress

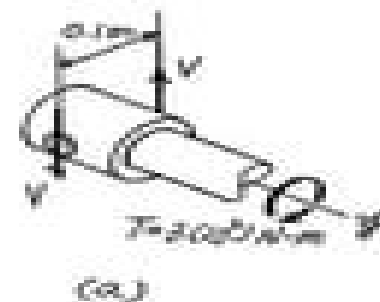
$$\tau_{\text{allow}} = \frac{\tau_{\text{fail}}}{FS} = \frac{150}{3} = 50 \text{ MPa}$$

Using this result,

$$\tau_{\text{allow}} = \frac{T}{A_s} \quad 50 \text{ MPa} = \frac{2.4 \text{ kN}\cdot\text{m}}{\frac{\pi}{4} d_s^2 L}$$

$$d_s = 0.02794 \text{ m} = 27.9 \text{ mm}$$

Ans.



8-80. Determine the maximum allowable torque T that can be transmitted by the joint. The shear pin A has a diameter of 25 mm, and it is made from a material having a failure shear stress of $\tau_{\text{fail}} = 150 \text{ MPa}$. Apply a factor of safety of 3 against failure.



$$\sum M_x = 0; \quad (150)(L) - T = 0$$

$$T = 150L$$

Allowable Shear Stress

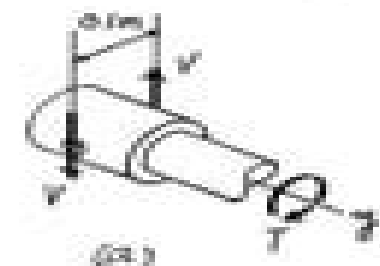
$$\tau_{\text{allow}} = \frac{\tau_{\text{fail}}}{FS} = \frac{150}{3} = 50 \text{ MPa}$$

The area of the shear plane for pin A is $A_s = \frac{\pi}{4} (25 \text{ mm})^2 = 490.874 \text{ mm}^2$. Using this result,

$$\tau_{\text{allow}} = \frac{T}{A_s} \quad 50 \text{ MPa} = \frac{150L}{490.874 \text{ mm}^2}$$

$$T = 1206.27 \text{ N}\cdot\text{m} = 1.21 \text{ kN}\cdot\text{m}$$

Ans.



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R. C. Hibbeler, S. C. Fan



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