$$\rho = \frac{m}{v} : \frac{kg}{vk^2} Density$$

$$V_{e} = \frac{1}{\rho} : \frac{m^{2}}{kg}$$
 Specific Volume

$$\gamma = \frac{W}{v} = \rho g; \frac{N}{m^3}$$
 Unit Weight

$$Gs = \frac{\rho_1}{\rho_8} = \frac{\rho_\theta}{\rho_8} = \frac{\gamma_s}{\gamma_w}$$
 Specific gravity water at 4°C  $\gamma = 62.5 \ lb/ft^2 = 9.81 \ kN/m^2$   $\rho = 1.94 \ slugs/ft^2 = 1000 \ kg/m^2$   $s = 1$ 

$$v = \frac{\mu}{\rho}$$
;  $m^2/s$  Kinetic Viscosity  
where /  
 $\mu = Pa$  · sec  
 $\rho = kg/m^2$   
note:  
I poise = 1 dyne ·  $s/cm^2$   
I stroke = 0.0001  $m^2/s$   
Idyne =  $10^{-2}N$ 

$$p = \frac{4\sigma}{d}, \frac{N}{m}$$
 Surface Tension  $\sigma$ 

$$h = \frac{4\sigma \cos \sigma}{5d} \text{ im Capillary}$$

$$h = \frac{4\sigma}{R^2}$$
 ;  $m$  Capillary (non circular)

$$h = \frac{2\sigma \cos \theta}{\rho gr}; m \ Capillary \ (tube)$$

$$l_g = \frac{bh^3}{36}$$
; Triangle

Flow rate

Q = Av:Volume flow rate

M = pQ: Mass flow rate

W = yQ: Weight flow rate

where

Q = discharge; m³/s

A = cross - sectional Area m²

v = velocity; m/s

p = density; kg/m³

y = weight density; N/m²

#### z: Elevation Head

### HYDRAULICS

$$E_B = \frac{\Delta P}{\Delta V}$$
: MPa Bulk Modulus

$$\beta = \frac{1}{E_{\pi}} : \frac{1}{MP\alpha} Compressibility$$

$$\rho = \frac{P}{RT} : Ideal Gas$$
where
$$P = absolute Pressure : Pa$$

$$R = J/kg *K$$

$$T = *K$$

$$R = 207/kg *K 1.716 ib fr/sing *R$$

$$*K = *C + 273 = *F + 460$$

$$\frac{p_1V_1}{T_1} = \frac{p_2V_2}{T_2}: New Ideal Gas$$

$$p_1V_1 = p_2V_2$$
: Isothermal

$$\frac{V_1}{T_1} = \frac{V_2}{T_2}$$
 : Isobaric

$$c=\sqrt{\frac{E_{o}}{\rho}}=\sqrt{\frac{1}{B_{\rho}}}; Pressure Disturbance$$

 $P_{abs} = P_{page} + P_{acm}$  Pressure abs  $P_{acm} = 101.325 \text{ kPa}$ 

$$p = \frac{F}{A}$$
; stress formula

$$h = \frac{p}{2}$$
, m Pressure Head

$$F = pA_1 N$$
 force on plane surface

 $F = pA_1 N$  force on plane surface

$$I_{\theta} = \frac{bh^2}{12}$$
 ; Triangle

# $\begin{array}{l} Q_1 = Q_2 = Q_3 \\ A_1v_1 = A_2v_2 = A_2v_3; Incompressible \\ \rho_1A_1v_1 = \rho_2A_2v_2 = \rho_3A_2v_2; Compressible \\ \gamma_1A_1v_1 = \gamma_2A_2v_2 = \gamma_2A_2v_2; Compressible \end{array}$

### $\frac{v^2}{2a}$ : Velocity Head

Pressure Head

$$E = \frac{v^2}{2g} + \frac{p}{\gamma} + z_1 Total Head$$

$$T = \frac{PD}{2}$$
: Tensile Force

$$S = \frac{PD}{2\sigma}$$
; Tensile (thickness)

 $P = y_{\omega}h_{z}$  Kpa Pressure

$$Bf = W : Bouyancy Force$$
  
 $\gamma V = W$   
 $Bf = W_{obj} + W_{applied} : Total BF$ 

$$\tan \theta = \frac{a}{\theta} : Horizontal$$

$$\tan \theta = \frac{a_h}{g \pm a_\nu}$$
: Inclined

$$p = \delta h \left(1 \pm \frac{\alpha}{a}\right)$$
; Vertical

#### Rotating Vessels

$$y = \frac{\omega r^2}{2g} : (radius)$$

$$y = \frac{\omega^2 x^2}{2g} : (x \text{ is given})$$
where
$$\omega = Angular \text{ velocity (rpm)}$$

$$y = height$$

$$note$$

$$1 \text{ rev} = 2\pi \text{ rad}$$

$$1 \frac{rad}{sec} = \frac{30}{\pi} \text{ rpm}$$

$$P = QyE.Power$$
  
 $\eta = \frac{Output}{Input} \times 100\% : Efficiency$   
note  
 $1 kp = 746 Watts = $50/t \cdot lb/sec$   
 $1 watt = 1 N \cdot m/s = 1 foules/sec$ 

$$RN = \frac{VD}{v}$$
; Reynolds Number Where  $V = Velocity$   $D = inside Diameter$   $v = kinetic Density$   $RN = \frac{\rho VD}{\mu}$  where  $\rho = density$   $\mu = dynamic viscosity$  note  $< 2100 - Laminar$   $> 2100 - turbulent$ 

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