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6-76. The joint is used to transmit a torque of $T = 2.4 \text{ kN}\cdot\text{m}$. Determine the required nominal diameter of the shaft pin A if it is made from a material having a shear failure stress of $\tau_{\text{fail}} = 150 \text{ MPa}$. Apply a factor of safety of 2 against failure.



Relevant Knowledge: The shear force developed on the shear plane of pin A can be determined by writing the moment equation of equilibrium along the x axis with reference to the free-body diagram of the shaft, Fig. a.

$$\sum M_x = 0; \quad (150)(L) - 500T = 0$$

$$L = 300 \text{ mm}$$

Allowable Shear Stress:

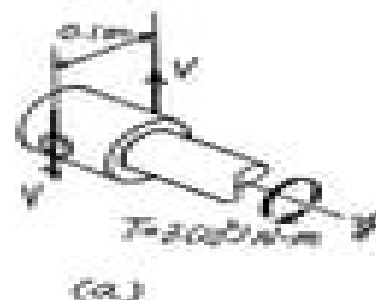
$$\tau_{\text{allow}} = \frac{\tau_{\text{fail}}}{FS} = \frac{150}{2} = 75 \text{ MPa}$$

Using this result,

$$\tau_{\text{allow}} = \frac{T}{J} \cdot \frac{D_p}{2} \quad 75 \text{ MPa} = \frac{500 \text{ N}\cdot\text{m}}{\frac{\pi}{32} D_p^4} \cdot \frac{D_p}{2}$$

$$D_p = 0.02764 \text{ m} = 27.6 \text{ mm}$$

Ans.



6-76. Determine the maximum allowable torque T that can be transmitted by the joint. The shear pin A has a diameter of 25 mm, and it is made from a material having a failure shear stress of $\tau_{\text{fail}} = 150 \text{ MPa}$. Apply a factor of safety of 2 against failure.



Relevant Knowledge: The shear force developed on the shear plane of pin A can be determined by writing the moment equation of equilibrium along the x axis with reference to the free-body diagram of the shaft, Fig. a.

$$\sum M_x = 0; \quad (150)(L) - T = 0$$

$$L = 300 \text{ mm}$$

Allowable Shear Stress:

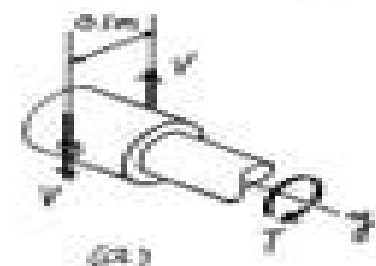
$$\tau_{\text{allow}} = \frac{\tau_{\text{fail}}}{FS} = \frac{150}{2} = 75 \text{ MPa}$$

The area of the shear plane for pin A is $A_s = \frac{\pi}{4} (0.025 \text{ m})^2 = 0.000491 \text{ m}^2$. Using this result,

$$\tau_{\text{allow}} = \frac{T}{A_s} \quad 75 \text{ MPa} = \frac{T}{0.000491 \text{ m}^2}$$

$$T = 3.69 \times 10^{-2} \text{ N}\cdot\text{m} = 3.69 \text{ kN}\cdot\text{m}$$

Ans.



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