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Initial Max. deformation

The maximum force $F = kx$ occurs when x is a maximum with $\dot{x} = 0$.

$$U_1 + \Delta T = mgs - \frac{1}{2}kx^2 = 0, \quad x = \frac{2mg}{k}$$

$$\text{So } F = kx = 2mg$$

$$3/129 \quad T_A + V_{A-0} = T_C : 0 + 2mgR = \frac{1}{2}mv_C^2, \quad v_C^2 = 4gR$$

$$(a) \quad \Sigma F_n = ma_n : N_B = m \frac{4gR}{R} = 4mg$$

$$(b) \quad T_A + V_{A-C} = T_C : 0 + 3mgR = \frac{1}{2}mv_C^2, \quad v_C^2 = 6gR$$

$$\Sigma F_n = ma_n : N_C - mg = m \frac{6gR}{R}$$

$$N_C = 7mg$$

(c) Coll. stopping point E:

$$T_A + V_{A-E} = T_E$$

$$0 + 2mgR - mg\left(\frac{1}{2}\right) - \mu_k \frac{15}{2}mg = 0$$

$$s = \frac{4R}{1 + \mu_k 15}$$

(Note: Normal force on incline is)

$$N = mg \cos 30^\circ = \frac{\sqrt{3}}{2}mg$$

3/130 Let s = distance down incline before reversal of direction.

$$U_1 + \Delta T = 100(2)(10 + s - s) - 200(10 + s - s)\frac{5}{13} = 1046 \text{ ft}\cdot\text{lb}$$

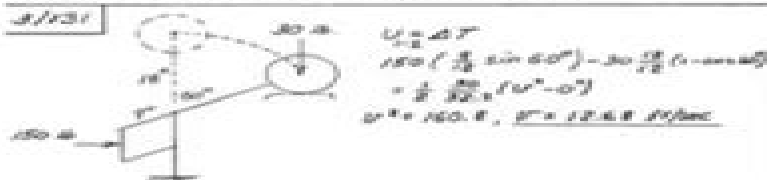
$$\Delta T = \frac{1}{2} \frac{350}{32.2} [v^2 - (17)^2] = 4.66v^2 - 377 \text{ ft}\cdot\text{lb}$$

$$U_1 + \Delta T : 1046 = 4.66v^2 - 377$$

$$v = 17.45 \text{ ft/sec}$$

The initial kinetic energy is positive regardless of the velocity direction.

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$$U_1 + \Delta T$$

$$150\left(\frac{1}{2} \sin 60^\circ\right) - 30 \frac{32.2}{16} (0 - v^2)$$

$$= \frac{1}{2} \frac{32.2}{16} (v^2 - 0)$$

$$v^2 = 160.8, \quad v = 12.68 \text{ m/sec}$$

$$3/132 \quad U_1 + \Delta T : mg(0.8 - 1.8 \cos 60^\circ)$$

$$= \frac{1}{2}m(12^2 - 9^2)$$

$$R(1)(0.20) = \frac{1}{2}(12^2 - 9^2), \quad 12^2 = 12.92, \quad 12 = 3.59 \text{ m/s}$$

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$$+\uparrow \Sigma F = 0 : 9810 - 4T = 0, \quad T = 2450 \text{ N}$$

Length of cable $L = 4s_0 + \text{constants}$

$$L = 4v_0 = 4(3) = 12 \text{ m/s}$$

$$P_{\text{out}} = -T\dot{L} = -2450(12) = 29400 \text{ watts}$$

$$\text{or } P_{\text{out}} = 29.4 \text{ kW}$$

$$C = \frac{P_{\text{out}}}{P_{\text{in}}}, \quad P_{\text{in}} = \frac{P_{\text{out}}}{C} = \frac{29.4}{0.8}$$

$$P_{\text{in}} = 36.8 \text{ kW}$$

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$$U_1 + \Delta T (\text{system})$$

$$\text{First interval: } 30(9.81)(1.2) - [30(9.81)(0.5 + 1.224)] \frac{1}{2} = \frac{1}{2} 30 v^2 + \frac{1}{2} 30 (10)^2$$

$$v^2 = 6.076 \text{ m/s}^2, \quad v = 2.447 \text{ m/s}$$

$$\text{Second interval: } 15(9.81)(2) - [30(9.81)(0.5 + 1.224)] \frac{1}{2} = 0 - \frac{1}{2} 15 (2.447)^2$$

$$- \frac{1}{2} 30 \left(\frac{5.076}{2}\right)^2$$

$$y = 2.18 \text{ m}, \quad s = \frac{1}{2}(1.2 + 2.18) = 1.69 \text{ m}$$

$$3/135 \quad U_1 + \Delta T : -\int_0^0 (3x^2 + 60x) dx = \frac{1}{2} \frac{400}{32.2} (0 - v^2) 12$$

$$x^2 + 20x^2 \Big|_0^0 = \frac{500}{32.2} v^2, \quad v \text{ in ft/sec.}$$

$$v^2 = \frac{32.2}{500} (60 + 480) = 60.82 (100 \text{ m/s})^2, \quad v = 240 \text{ ft/sec}$$

$$3/136 \quad \Theta = \tan^{-1} \frac{6}{100} = 3.43^\circ$$

$$U_1 + \Delta T : U_1 + mgh = \frac{1}{2} m (v_1^2 + v_2^2)$$

$$U_1 = -1400(9.81)(200 \sin 3.43^\circ)$$

$$+ \frac{1}{2} 1400 \left[\left(\frac{20}{5.28}\right)^2 - \left(\frac{100}{5.28}\right)^2 \right]$$

$$= -683,000 \text{ J} \quad \text{or} \quad -683 \text{ kJ}$$

$$\text{Energy lost } Q = 683 \text{ kJ}$$

3/137 The power output of the drivetrain is

$$P_{\text{out}} = Fv = 540 \left(\frac{30}{3.6}\right) = 14,400 \text{ W}$$

The power input to the drivetrain:

$$P_{\text{in}} = \frac{P_{\text{out}}}{\epsilon} = \frac{14,400}{0.70} = 20,571 \text{ W}$$

$$\text{So the motor output } P = 20.6 \text{ kW}$$

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**James L. Meriam, L. G. Kraige, J. N.
Bolton**



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Principles of Vibration Analysis with Applications in Automotive Engineering Ronald L Huston, C Q Liu, 2011-01-10 This book written for practicing engineers designers researchers and students summarizes basic vibration theory and established methods for analyzing vibrations Principles of Vibration Analysis goes beyond most other texts on this subject as it integrates the advances of modern modal analysis experimental testing and numerical analysis with fundamental theory No other book brings all of these topics together under one cover The authors have compiled these topics compared them and provided experience with practical application This must have book is a comprehensive resource that the practitioner will reference

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Essentials of Dynamics and Vibrations John Billingsley,2025-03-02 This updated second edition brings the complex mathematics of three dimensional dynamics to life with real time simulations making the equations easier to grasp Covering core topics in mechanical engineering such as kinematics dynamics vibration analysis gyroscopes gears and Euler s equations the book offers a clear and engaging approach for students professionals and enthusiasts alike With a focus on practical applications it explains everything from the laws of motion to motors and mechanisms providing a comprehensive understanding of mechanical systems New to this edition is a chapter on Power Energy and Perpetual Motion which reveals intriguing comparisons such as the energy needed to lift water versus the heat required to warm it The final chapter Rocket Science has been expanded to debunk myths about black holes and gravity humorously addressing science fiction misconceptions while proposing exciting space projects *800 Solved Problems in Vector Mechanics for Engineers* Joseph

F. Shelley,1990 **Solving Dynamics Problems in Maple by Brian Harper T/a Engineering Mechanics Dynamics 6th Edition by Meriam and Kraige** Brian D. Harper,J. L. Meriam,L. G. Kraige,2006-12-15 *Solving Dynamics Problems in MathCad A Supplement to Accompany Engineering Mechanics: Dynamics, 5th Edition by Meriam & Kraige* Brian Harper,2001-11-26 If MathCad is the computer algebra system you need to use for your engineering calculations and graphical output Harper s *Solving Dynamics Problems in MathCad* is the reference that will be a valuable tutorial for your studies Written as a guidebook for students taking the Engineering Mechanics course it will help you with your engineering assignments throughout the course Over the past 50 years Meriam Kraige s *Engineering Mechanics Dynamics* has

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