

## EXAMPLE 8.7

The turnbuckle shown in Fig. 8-17 has a square thread with a mean diameter of 1 mm and a lead of 1 mm. If the coefficient of static friction between the wire and the turnbuckle is  $\mu_s = 0.25$ , determine the moment  $M$  that must be applied to draw the end screws closer together.



Fig. 8-17



## SOLUTION

The moment can be obtained by applying Eq. 8-3. Since friction at two corners must be considered, this requires

$$M = 2rW \tan \phi = dW \quad (1)$$

Here  $W = 2000 \text{ N}$ ,  $d = \tan^{-1} \mu_s = \tan^{-1}(0.25) = 14.04^\circ$ ,  $r = 1 \text{ mm}$ , and  $\theta = \tan^{-1}(1/2\pi) = \tan^{-1}(2 \text{ mm}/31.415 \text{ mm}) = 3.64^\circ$ . Substituting these values into Eq. 1 and solving gives

$$\begin{aligned} M &= 2(2000 \text{ N})(1 \text{ mm}) \tan 14.04^\circ + 3.64^\circ \\ &= 874.7 \text{ N}\cdot\text{mm} = 0.875 \text{ N}\cdot\text{m} \end{aligned} \quad (2)$$

**NOTE:** When the moment is removed, the turnbuckle will be self-locking, i.e., it will not unscrew since  $d_s > \theta$ .

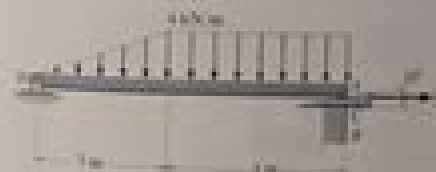
## PROBLEMS

**8-38.** Determine the largest angle  $\theta$  that will cause the wedge to be self-locking regardless of the magnitude of horizontal force  $P$  applied to the blocks. The coefficient of static friction between the wedge and the blocks is  $\mu_s = 0.3$ . Neglect the weight of the wedge.



Prob. 8-38

**8-39.** If the beam  $AB$  is loaded as shown, determine the horizontal force  $P$  which must be applied to the wedge in order to remove it from under the beam. The coefficient of static friction at the wedge's top and bottom surfaces are  $\mu_{s1} = 0.25$  and  $\mu_{s2} = 0.35$ , respectively. If  $P = 0$ , is the wedge self-locking? Neglect the weight and size of the wedge and the thickness of the beam.



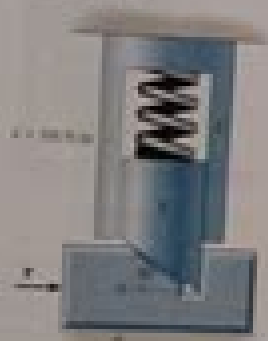
Prob. 8-39

**8-40.** The wedge has a negligible weight and a coefficient of static friction  $\mu_s = 0.35$  with all contacting surfaces. Determine the largest angle  $\theta$  so that it is self-locking. The response for slipping for any magnitude of the force  $P$  applied to the pins.



Prob. 8-40

**8-41.** If the spring is compressed 40 mm and the coefficient of static friction between the inclined rail  $E$  and the slider is  $\mu_{s1} = 0.3$ , determine the horizontal force  $P$  needed to move the slider forward. The rail is free to move within friction rollers at the fixed roller  $F$ . The coefficient of static friction between  $A$  and surface  $B$  is  $\mu_{s2} = 0.4$ . Neglect the weights of the slider and rail.



Prob. 8-41

**8-42.** If  $P = 20 \text{ N}$ , determine the required minimum compression in the spring so that the wedge will not move to the right. Neglect the weights of  $A$  and  $B$ . The coefficient of static friction for all contacting surfaces is  $\mu_{s1} = 0.35$ . Neglect friction at the rollers.

**8-43.** Determine the minimum applied force  $P$  required to move wedge  $A$  to the right. The spring is compressed a distance of 175 mm. Neglect the weights of  $A$  and  $B$ . The coefficient of static friction for all contacting surfaces is  $\mu_s = 0.35$ . Neglect friction at the rollers.



Prob. 8-43

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