Formula Sheet

Stresses
$$\sigma = \frac{P}{A} \qquad \sigma = \frac{M \cdot c}{I} \qquad \tau = \frac{T \cdot c}{J}$$
 Unsymmetric bending
$$\sigma = \frac{P}{A} + \frac{-M_z y}{I_z} + \frac{-M_y z}{I_y} \qquad \tan \phi = \frac{I_z}{I_y} \tan \theta$$
 Shear stress
$$\tau = \frac{VQ}{R} \qquad q = \tau \cdot t = \frac{VQ}{I}$$
 Thin-walled pressure vessel:
$$\sigma_1 = \frac{P^r}{A} \qquad \sigma_2 = \frac{P^r}{A}$$

Stress Transformation

$$\begin{split} \sigma_{c'} &= \frac{\sigma_{c} + \sigma_{y}}{2} + \frac{\sigma_{c} - \sigma_{y}}{2} \cos 2\theta + \tau_{cy} \sin 2\theta & \tan 2\theta_{y} = \frac{2 \tau_{cy}}{(\sigma_{c} - \sigma_{y})} \\ \tau_{c'y'} &= -\frac{\sigma_{c} - \sigma_{y}}{2} \sin 2\theta + \tau_{cy} \cos 2\theta & \tan 2\theta_{y} = -\frac{(\sigma_{c} - \sigma_{y})}{2 \cdot \tau_{cy}} \\ \tau_{max} &= \sqrt{\left(\frac{\sigma_{c} - \sigma_{y}}{2}\right)^{2} + \tau_{cy}^{2}} & \sigma_{mg} = \frac{\sigma_{c} + \sigma_{y}}{2} \\ \sigma_{L2} &= \frac{\sigma_{c} + \sigma_{y}}{2} \pm \sqrt{\left(\frac{\sigma_{c} - \sigma_{y}}{2}\right)^{2} + \tau_{cy}^{2}} \end{split}$$

Beam Design:
$$S_{\text{min}} = \frac{|M|_{\text{met}}}{\sigma_{\text{eff}}}$$
 $r_{\text{m}} = \frac{V_{\text{max}}}{A_{\text{mod}}}$, where $A_{\text{mod}} = d \cdot t_{\text{mod}}$

Yield Criteria

Von Mises (Distortion Energy):
$$\sigma_a^2 - \sigma_a \sigma_b + \sigma_b^2 < \sigma_T^2$$

Tresca (Maximal Shear Stress):
$$|\sigma_s| < \sigma_T$$
 and $|\sigma_b| < \sigma_T$
or $|\sigma_s - \sigma_b| < \sigma_T$

Strain Gauge Transformation
$$\varepsilon(\theta) = \varepsilon_{\rm c} \cos^2 \theta + \varepsilon_{\rm p} \sin^2 \theta + \gamma_{\rm qp} \sin \theta \cos \theta$$

Generalized Hook's Law

$$\varepsilon_{x} = \frac{1}{\mathcal{B}} \left[\sigma_{x} - v(\sigma_{y} + \sigma_{z}) \right] \qquad \qquad \gamma_{yz} = \frac{1}{G} \tau_{yz}, \qquad \gamma_{zz} = \frac{1}{G} \tau_{zz}, \qquad \gamma_{zz} = \frac{1}{G} \tau_$$

Mechanical Engineering Formulas Cheat Sheet

Michael Brown

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